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DESIGN AND ANALYSIS OF THE COMPOSITE SPUR GEAR

Anuj Nath¹, A.R. Nayak²

¹ M.Tech Student, ²Assistant Professor, Mechanical Engineering, Swamy Vivekananda Engineering College, Bobbili A.P (India)

ABSTRACT

To design the spur gear to study the weight reduction and stress distribution of cast steel and composited material. Gearing is one of the most critical components in the mechanical power transmission system and in the most industrial rotating machinery. It is possible that gears will predominate as the most effective means of transmitting power in future machines due to their high degree of reliability and compactness. In addition, the rapid shift in industry from heavy industry such as ship building to industries such as auto mobile manufacture and auto machine tools will necessitate a refined application of gear technology. To design the spur gear model using the PRO-E software. To study the impact analysis for cast steel and composite materials. To study the torque loading for cast steel and composite materials. Finally, comparing and analyzing the composite gears with existing cast steel gear is to be done by using ANSYS 13.0.

Keywords: The Spur Gear, Torque Loading for Cast Steel and Composite Materials

I. INTRODUCTION

1.1 Gear Technology

Gears are the most common means of transmitting power in the modern mechanical engineering world. They vary from a tiny size used in watches to the Large gears used in watches to the large gears used in lifting mechanisms and speed reducers. They form vital elements of main and ancillary mechanisms in many machines such as automobiles, tractors, metal cutting machine tools etc. Toothed gears are used to change the speed and power ratio as well as direction between input and output[1].

1.2 Spur Gear

The spur gear is simplest type of gear manufactured and generally used for transmission of rotary motion between parallel shafts. The spur gear is the first choice option for gears except when high speeds, loads and ratios direct towards other options. Other gear types may also be preferred to provide more silent low-vibration operation. A single spur gear is generally selected to have a ratio range of between 1:1 and 1:6 with a pitch line velocity up to 25 m/s [6]. The spur g ear has an operating efficiency of 98-99%. The pinion is made from a harder material than the wheel.

A gear pair should be selected to have the highest number of teeth consistent with a suitable safety margin in strength and wear. The minimum number of teeth on a gear with a normal pressure angle of 20 degrees is 18. This is a cylindrical shaped gear in which the teeth are parallel to the axis. It has the largest applications and it is the easiest to manufacture.

They are simple in construction, easy to manufacture cost less. They have highest efficiency and excellent precision rating. They are used in high speed and high load application in all types of trains and a wide range of

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velocity ratios. Hence, They find wide applications right from clocks, household gadgets, motor cycles, Automobiles and railways to aircrafts.



Figure 1.3 Spur Gear

1.3. Gear Design Procedure

Following are the Formulas used in Gear Design

- Module = m
- Face width = b
- Number of teeth on pinion =Z1
- Number of teeth on gear = Z2
- Speed of pinion = N1
- Speed of gear = N2
- ✤ Gear ratio or Transmission ratio = v
- v = Z2 / Z1

1.4 Design of Spur Gear

Calculations

TORQUE (**T**) = 13.8kg-m@2500rpm

- ◆ T = 13.8 kg-m
- ♦ T = 13.8*10 N-m
- ✤ T = 138 N-m
- ✤ T = 138000 N-mm
- ♦ N = 2500 rpm.
- **♦ POWER (P)** = 2*3.14*2500*T/60
- $\bullet \quad P = 2*3.14*2500*138/60$
- ◆ P = 36128 Watt
- ◆ Power (P) = 36.128 K Watt.
- Torque (T) = $F^*(d/2)$
- Where,
- ✤ F-load,
- d- Pitch circle diameter (z*m=180mm)T = F*(d/2)
- F = T/(d/2)
- ✤ F = 138000/90
- ✤ Load (F) = 1533.33N

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- Using Lewis equation,
- Tangential load, $F = b^*y^*pc^*\sigma b$
- Pc = 3.14* module
- ◆ Pc = 31.4mm
- ◆ Y= Lewis form factor=0.134mm
- ***** $\quad b = face width = 54mm$
- The maximum allowable stress= 8.7413 N/mm2.
- Ultimate tensile strength for cast steel = 540mpa Ultimate tensile strength for composite = 52mpa
- Allowable stress for cast steel = ultimate tensile strength/3
- $\ \, \bigstar \ \ \, = 540/3 = 180 N/mm2 > 8.7413 N/mm2 \\$
- Allowable stress for composite = ultimate tensile strength/3
- ◆ = 52/3 = 17.33N/mm2>8.7413N/mm2
- So, the design is safe.

II. CALCULATIONS OF GEAR TOOTH PROPERTIES

Pitch circle diameter (p.c.d) = $z^*m = 18^*10 = 180mm$

- Base circle diameter (Db) = $D \cos \alpha$
- $= 180 * \cos 20$
- = 169.145mm

Outside circle diameter = (z+2)*m=(18+2)*10 = 200mm

Clearance = circular pitch/20 = 31.4/20 = 1.57mm

Dedendum = Addendum + Clearance = 10 + 1.57 = 11.57mm

- Module = D/Z = 180/18 = 10mm
- Dedendum circle diameter = P.C.D 2* dedendum
- ◆ = 80-2*11.57= 156.86mm
- Fillet radius = Circular pitch/8 = 31.4/8 = 3.9mm
- Pitch circle diameter (Pc) = $m^*z = 10^*18 = 180$ mm
- Hole depth = 2.25*m = 2.25*10 = 22.5mm
- Thickness of the tooth = 1.571*10 = 15.71mm
- Face width (b) = 0.3*180=54mm
- Center distance between two gears = 180mm
- Diametral pitch = Number of teeth/P.C.D= 18/180= 0.1mm

III. PROPERTIES OF CAST STEEL

- ✓ Density = 7870 kg/m3
- ✓ Young modulus = 200 GPa
- ✓ Poisson's ratio = 0.29
- \checkmark Tensile strength = 518.8 MPa
- ✓ Ultimate Tensile Strength = 540 MPa
- ✓ Yield Tensile Strength = 415 MPa
- ✓ Bulk modulus = 140 GPa

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IV. COMPOSITE MATERIALS

A composite material can be defined as a combination of two or more materials that results in better properties than those of the individual components used alone. In contrast to metallic alloys, each material retains its separate chemical, physical, and mechanical properties.

The two constituents are reinforcement and a matrix. The main advantages of composite materials are their high strength and stiffness, combined with low density, when compared with bulk materials, allowing for a weight reduction in the finished part.

The reinforcing phase provides the strength and stiffness. In most cases, the reinforcement is harder, stronger, and stiffer than the matrix. The reinforcement is usually a fiber or a particulate.

properties of composites (50% carbon fibers in epoxy resin matrix)

- **b** Density = 1800 kg/m
- Young modulus = 450 G Pa
- Poisson's ratio = 0.30
- Tensile strength = 52 M Pa



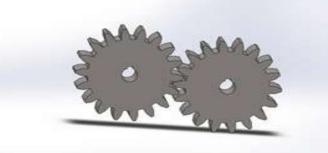


FIG 4.3 Spur Gear Models

V. TORQUES. RESULTS AND DISCUSSION

5.1 Analysis Results for Spur Gear in Various Materials
5.1.1 Reports for Cast Steel Spur Gear in Various
TORQUE T = 140N-m; SPEED N = 2500 rpm

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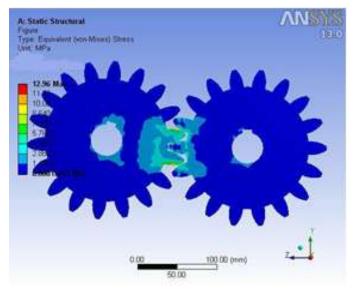


Fig 5.1 Von-Mises Stress Distribution of Spur Gear in Cast steel

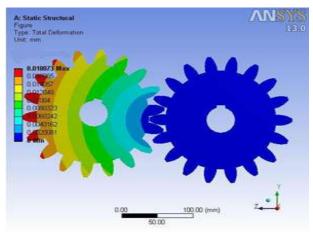


Fig 5.3 Total Deformation of Spur Gear in Cast steel

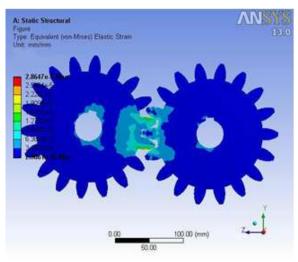


Fig 5.6 Von-Mises Stress Distribution of Spur Gear in Composite materials

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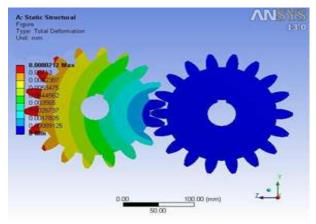


Fig 5.8 Total Deformation of Spur Gear in Composite materials

5.2 Comparison Table between Cast Steel and Composite Materials

| SPEED | CAST STEEL | | | COMPOSITE MATERIALS | | | |
|--------------|--|---|---|---|---|---|---|
| AND | | | | | | | %DIFF |
| TORQUE | | | | | | | ERIAN |
| | | | | | | | CE |
| SPPED | 2500 | 2000 | 1500 | 2500 | 2000 | 1500 | |
| (IN RPM) | RPM | RPM | RPM | RPM | RPM | RPM | |
| TORQUE | 140 | 170 | 230 | 140 | 170 | 230 | |
| (Nm) | Nm | Nm | Nm | Nm | Nm | Nm | |
| (MPa) | 12.960 | 15.737 | 21.292 | 12.891 | 15.654 | 21.179 | 0.5324 |
| l | 6.48 e-5 | 7.868 e-5 | 10.646 e-5 | 2.865 e-5 | 3.479 e-5 | 4.706 e-5 | 55.787 |
| n(MPa) | 18.073 e-3 | 21.945 e-3 | 29.691e-3 | 8.021e-3 | 9.740 e-9 | 13.178 e-9 | 55.619 |
| stress (MPa) | 7.376 | 8.956 | 12.117 | 7.342 | 8.915 | 12.062 | 0.4610 |
| J) | 157.87 e-3 | 232.78 e-3 | 426.09 e-3 | 69.889 e-9 | 103.05 e-3 | 188.62 e-3 | 55.730 |
| | AND TORQUE SPPED (IN RPM) TORQUE (Nm) G(MPa) G(MPa) Stress (MPa) | AND TORQUE2500SPPED2500(IN RPM)RPMTORQUE140(Nm)NmG (MPa)12.960G (MPa)18.073 e-3stress (MPa)7.376 | AND TORQUE 2500 2000 SPPED 2500 2000 (IN RPM) RPM RPM TORQUE 140 170 (Nm) Nm Nm G(MPa) 12.960 15.737 m(MPa) 18.073 e-3 21.945 e-3 stress (MPa) 7.376 8.956 | AND TORQUE 2500 2000 1500 SPPED 2500 2000 RPM (IN RPM) RPM RPM RPM TORQUE 140 170 230 (Nm) Nm Nm Nm G(MPa) 12.960 15.737 21.292 n(MPa) 18.073 e-3 21.945 e-3 29.691e-3 stress (MPa) 7.376 8.956 12.117 | AND TORQUE SPPED 2500 2000 1500 2500 SPPED 2500 RPM RPM RPM RPM TORQUE 140 170 230 140 Nm Nm Nm Nm Nm G(MPa) 12.960 15.737 21.292 12.891 m(MPa) 18.073 e-3 21.945 e-3 29.691e-3 8.021e-3 stress (MPa) 7.376 8.956 12.117 7.342 | AND TORQUE SPPED 2500 2000 1500 2500 2000 (IN RPM) RPM RPM RPM RPM RPM RPM RPM TORQUE 140 170 230 140 170 (Nm) Nm Nm Nm Nm Nm (MPa) 12.960 15.737 21.292 12.891 15.654 n(MPa) 18.073 e-3 21.945 e-3 29.691e-3 8.021e-3 9.740 e-9 stress (MPa) 7.376 8.956 12.117 7.342 8.915 | AND TORQUE SPPED 2500 2000 1500 2500 2000 1500 SPPED 2500 2000 RPM R |

IV. CONCLUSION

- * The literature survey of composite spur gear was performed. Then the study in weightreduction and stress distribution of spur gear for cast steel and composite materials has been done.
- * On the basis of that study, the analysis of both cast steel and composite materials are analyzed in the application of gear box which is used in automobile vehicles. From these analysis we got the stress values for composite materials is less as compared to the cast steel spur gear.
- So from these analysis results, we conclude that, the stress induced, deformation and weight of the * composite spur gear is less as compared to the cast steel spur gear.
- So, Composite materials are capable of using in automobile vehicle gear boxes up to 1.5KN in the application of Tata super ace model instead of existing cast steel gears with better results.

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VII. SCOPE FOR FUTURE WORK

- ♦ Various composite materials can be applied instead of currently used materials.
- The input conditions can be varied to parameters like pressure, temperature etc.
- ✤ A study on wear, friction and temperature effects can be extended

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