

DESIGN OF MONO LEAF SPRING USING CAE TOOLS WITH COMPOSITE MATERIAL

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ABSTRACT

Weight reduction is now the main issue in automobile industries. Weight reduction can be achieved primarily by the introduction of better material, design optimization and better manufacturing processes. The suspension leaf spring is one of the potential items for weight reduction in automobile as it accounts for ten to twenty percent of the unsprung weight, which is considered to be the mass not supported by the leaf spring. Since the composite materials have more elastic strain energy storage capacity and high strength-to-weight ratio as compared to those of steel, Composite material is taken into consideration. The objective of present work is to minimize the deflection in the leaf spring and reduce the weight by changing the parameters i.e. size & shape. In the work the model of leaf spring is designed in CATIA V5 software and saved in iges or stp format. The model is then imported in Solid Works software and saved in Para solid (x_t) format file for no data loss. This Para solid file of leaf spring is then imported in ANSYS workbench. The deformation and stress contours have been plotted and the results obtained are compared with available results in literature survey. All these will result in fuel saving.

Keywords: ANSYS, CATIA V5, Composite Material, FEA, Leaf Spring

I. INTRODUCTION

A leaf spring is a simple form of spring commonly used for the suspension in wheeled vehicles. Originally called a laminated or carriage spring, and sometimes referred to as a semi-elliptical spring or cart spring. Leaf springs are mainly used in suspension systems to absorb shock loads in automobiles like light motor vehicles, heavy duty trucks and in rail systems. It carries lateral loads, brake torque, driving torque in addition to shock absorbing . A composite material is the combination of two or more materials that produce a synergistic effect so that the combination produces aggregate properties that are different from any of those of its constituents attain independently. These two or more materials can be mixed in a restricted way to achieve optimum properties, which are superior to the properties of each individual component or material. Composite materials have been widely used in automobile industry because of its high strength and modulus to weight ratio, low cost and flexibility in material and structure design.[6,7] Since the strain energy in the spring is inversely proportional to density and young's modulus of the material, it is always suggested that the material for leaf spring must have low density and modulus of elasticity.

Weight reduction has been the main focus of automobile manufacturers in the present scenario. The replacement of steel with optimally designed composite leaf spring can provide 92% weight reduction.[3] Moreover the composite leaf spring has lower stresses compared to steel spring. According to the studies made a material with maximum strength and minimum modulus of elasticity in the longitudinal direction is the most suitable material for a leaf spring. Fortunately, composites have these characteristics. The Arrangement of Leaf Spring shown in Figure 1.

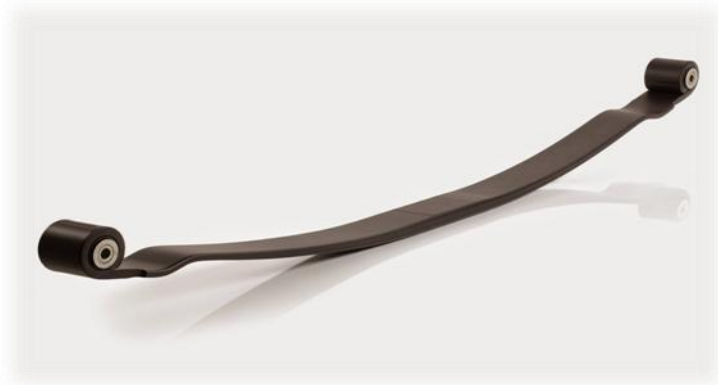


Fig. 1 Mono Leaf Spring

II. LITERATURE

Pradhan A. et al. (2016) performed the comparison of performance of various leaf spring material used in automobile industry. This work carry the modeling & analysis of the conventional steel (55Si2Mn90) & three composite leaf springs with the same loading & boundary condition.[3]

K. Kumar et al. (2015) optimized a mono parabolic leaf spring using CAE software. In this work It was shown that the use of composite material instead of steel resulted into large deflection, small variation in stresses and also a large amount of weight reduction [4].

Ashvini P. Lad. et al. (2015) performed the deflection analysis of steel leaf spring vs composite leaf spring through FEA software. The work carried out in this paper is to present analysis of deflection of composite leaf spring (Epoxy Carbon fiber) to the conventional steel leaf spring through FEA software ie in Ansys.[7]

Manjunath H.N et al. (2014) carried the static analysis & fatigue life prediction of composite leaf spring for a light commercial vehicle. In this research work an attempt has been made to check the suitability of composite materials like E Glass/Epoxy, Graphite/Epoxy, Boron/Al, Carbon/ Epoxy & Kevlar/ Epoxy for light commercial vehicle leaf spring.[6]

Ghodake A. P. et al. performed analysis of steel & composite leaf spring for vehicle. the present study searched about new composite material.[2]

III. METHODOLOGY

There are three steps in software based Finite Element Analysis-

- 3.1 Pre-processing
- 3.2 Analysis
- 3.3 Post processing

Figure 2 shows typical FEA procedures by commercial software.[2]

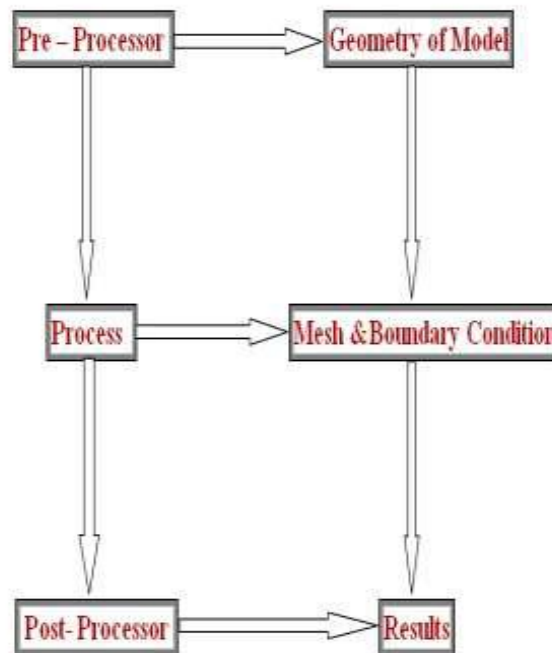


Fig.2 Typical FEA procedures by commercial software

3.1 Pre-Processing

Prepare Model: The first step is to prepare a CAD model of master leaf spring. The model of leaf spring is designed in CATIA V5 software and saved in iges or stp format. The geometrical specification of leaf spring is given in table 1[4].

Table 1 Specification of Leaf Spring

Span Length	940 mm
Number of Leaf	1
Rated Load	1500 N
Width of Leaf	60 mm
Centre Rubber Pad	100 mm X 50 mm X 5mm

The model of leaf spring is then imported in Solid Works software and saved in Para solid (x_t) format file for no data loss. This Parasolid file of leaf spring is then imported in ANSYS workbench. In the present work,

material is removed from the bottom of leaf spring up to thickness 1 mm as shown in Figure 3. The deformation and stress contours have find out using ANSYS workbench. The results obtained are compared with available results in literature survey.

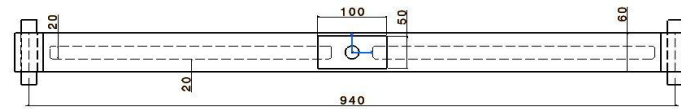


Fig. 3 Leaf Spring Drawing after Modification

The Fig 4 shows the CAD model of leaf spring in CATIA V5 Software.

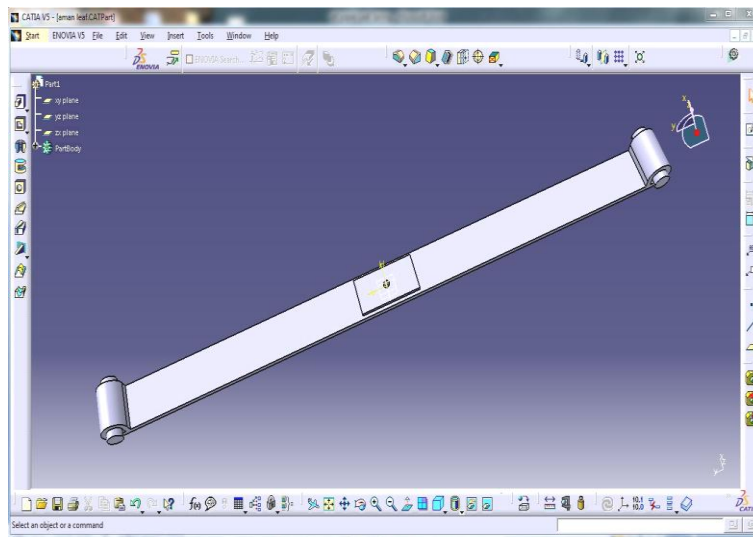


Fig. 4 CAD model of leaf spring

Mesh generation: Second step is to generate mesh using parabolic tetrahedral elements. An automatic method is used to generate the mesh in the present work. Fig 5 shows the meshed model of leaf spring in AYSYS workbench.

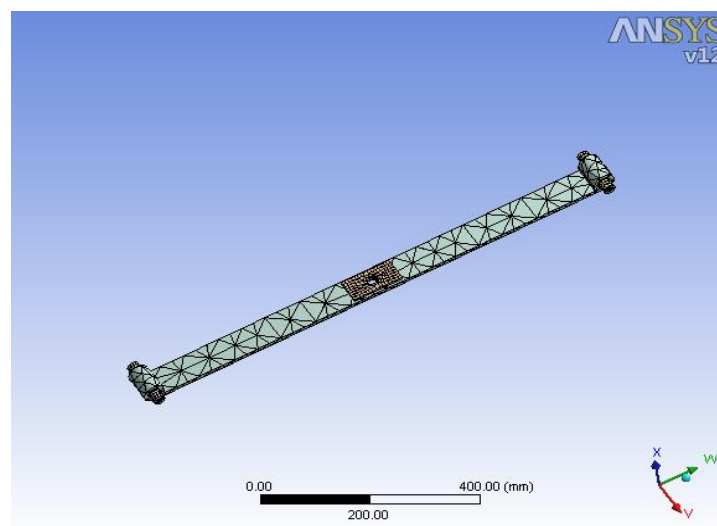


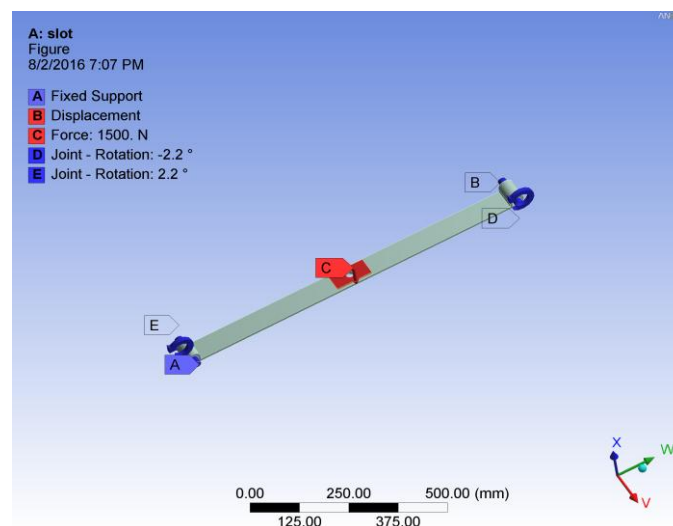
Fig. 5 Meshed model of Leaf Spring in ANSYS workbench

3.2 Analysis

Boundary Conditions: Third step is to apply boundary conditions. The boundary condition is the collection of different forces, supports, constraints and any other condition required for complete analysis. Applying boundary condition is one of the most typical processes of analysis. A special care is required while assigning loads and constraints to the elements. Boundary condition of the leaf spring involves the fixation of one of the revolute joint and applying displacement support at the other eye end of leaf spring. A joint rotation of 2.2° has been taken for both revolute joints considering the no load camber. Loading conditions involves applying a load of 1500 N at the centre of the leaf. As per specifications the spring is drawn at flat condition, therefore the load is applied in downward direction to achieve initial no load condition. The model under defined boundary conditions is shown in Fig 6[4].

Model Display: While applying the boundary conditions, it is necessary to view the model from different angles Pre-Processor offers capabilities of rotating, smoothness, scaling, regions, active set, etc. for efficient model viewing and editing.

Solution: The Solution phase deals with the solution of the problem according to the problem definitions. All the tedious type of work of formulating assembling of matrices is done by the computer and finally displacements and stress values are given as output.

**Fig. 6 Applied constraints on the Leaf Spring**

3.3 Post- Processor:

It is a powerful user friendly Post-Processing program using interactive colour graphics. It has extensive plotting features for displaying the results obtained from the Finite Element Analysis. One picture of the analysis results (i.e. the results in a visual form) can often reveal in seconds what would take an engineer hour to access from a numerical output, say in tabular form. The engineer may also

see the important aspects of the results that could be easily missed in a stack of numerical data. The entire range of Post-Processing options of different types of analysis can be accessed through the command/menu mode there by giving the user added flexibility convenience Employing state of art image enhancement techniques facilitates the viewing of:

- Contours of stresses, displacements, temperatures etc,
- Deform geometric plots, light source shaded plot,
- Animated deformed shapes,
- Time-history plots,
- Solid sectioning hidden line plot and boundary line plot etc.

IV. STATIC STRUCTURAL ANALYSIS

Static structural analysis is done in ANSYS workbench to find out the equivalent (von-mises) stress and total deformation. Fig.7 and Fig. 8 show equivalent (von-mises) stress and total deformation in the leaf spring.

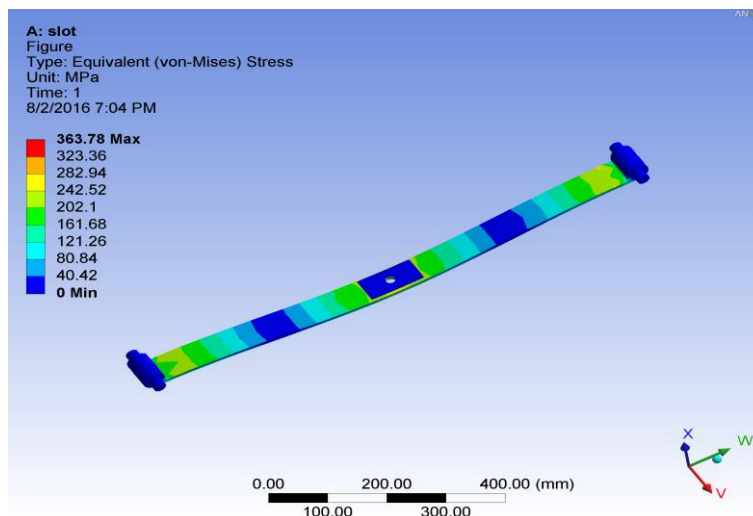


Fig. 7 Equivalent (von-mises) stress

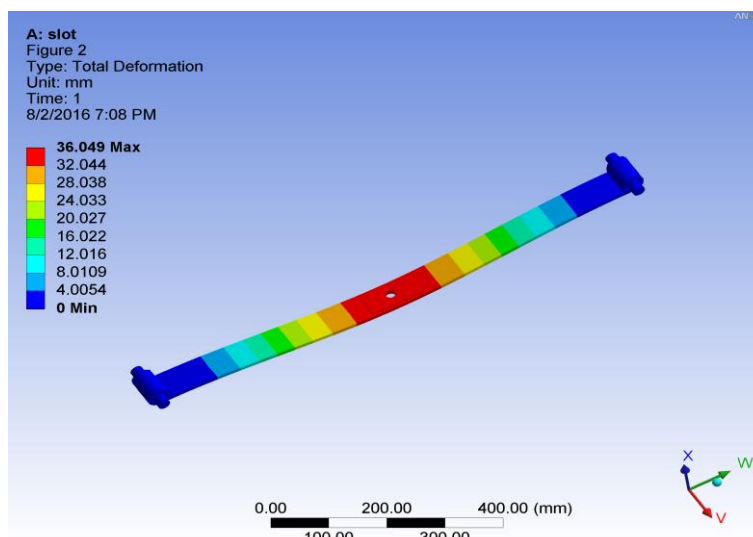


Fig. 8 Total Deformation

From the below comparison Table 2 it has been observed that for the same static load and boundary conditions, the deflection in the case of existing leaf is 71.724 mm while in case of proposed leaf it is 36.049 mm which shows that proposed leaf has less deflection in comparison to existing one. Also the value of von-mises stress has been decreased from the 501.26 MPa to 363.78 MPa. At the same time a weight also reduces from the 1.3716 kg to 1.0742 kg.

Table 2 FEA results of Leaf Spring

Sr. No.	Parameters	Existing Design	Proposed Design
1.	Normal Static Load	1500 N	1500 N
2.	Equivalent (von-mises) stress	501.26 MPa	363.78 MPa
3.	Total deformation in mm	71.724 mm	36.049 mm
4.	Total Weight	1.3716 kg	1.0742 kg

V. CONCLUSION

Finite Element Analysis of the leaf spring has been done using ANSYS Workbench. From the results obtained from FE Analysis, many discussions have been made. The results obtained are well in agreement with the available existing results. The model presented here, is well safe and under permissible limit of stresses.

1. On the basis of the current work, it is concluded that the proposed material give sufficient improvement in the existing results.
2. The weight of the leaf spring is also reduced by 21.6 %, thereby reducing the cost.
3. The stress is found maximum near the hole and sharp edges.

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