

STATIC STRESS ANALYSIS OF DOMESTIC WIND

MILL BLADE

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ABSTRACT

There are two primary physical principles by which energy can be extracted from the wind; these are through the creation of either lift or drag force (or through a combination of the two). Drag forces provide the most obvious means of propulsion, these being the forces felt by a person (or object) exposed to the wind. Lift forces are the most efficient means of propulsion but being more subtle than drag forces are not so well understood. So the static stress analysis of domestic wind mill blade. The wind force is developed by the wind on the blade surface .due to this the stresses is developed in the blade

Keywords: blade, E glass epoxy, FEA method , load ,static stress, wind mill.

I. INTRODUCTION

The FE method has been used comprehensive design the past decades. It is a very useful method which captures all the necessary details and allows the designer to determine structure responses for a variety of load case. FE models of wind turbine blades normally use layered shell elements. These elements provide an efficient means of doweling structures composed of laminate composite materials which is used in the wind turbine industry. The orthogonal stiffness properties of the elements are calculated by the use of laminate theory. In this study two different types of 3D shell elements are used; a nonlinear laminated shell element for modeling the sandwich parts and a near laminated shell element for modeling the rest. The types of loading that are applied in the model include an externally applied force due to the wind which is known as aerodynamic load & Inertial forces. Aerodynamic loads are obtained using the Blade Element Momentum (BEM) theory. Lift and drag forces generated in steady wind conditions are analyzed as normal and tangential forces and the blade sections. These forces are applied as boundary loads on some specific nodes on the FE model and along the blade. Providing rotational velocity and gravitational acceleration by the user, inertial forces are applied automatically by ANSYS that is used in this simulation and are combined with the mass matrices to form a body force load vector term. However, inertia loads are effective only if the model has some mass. To do so, density specification is used this research.

II.THEORETICAL ANALYSIS

The wind mill blade is fixed at one end and free at another end that is cantilever type. The load is applied at the centre of the blade. Due to this the bending stresses is developed. They are calculated as follows

Length of the blade = 900mm



Width of the blade (B)= 74mm

Thickness of the blade (T) =13mm

The moment due to applied load at the center of the blade = $M = \frac{WL}{4}$

Where W=applied load

L= length of blade

Section Modulus of blade section (Z) = $\frac{I}{Y} = \frac{BT^2}{12} = \frac{74 \times 13^2}{12} = 1042.16 \text{mm}^3$

The bending stress developed is calculated by equation $\frac{M}{I} = \frac{F}{Y}$ i.e. $f = \frac{M}{Z}$

Table 1: Theoretical Static Stress Analysis Of E Glass Epoxy Blade

Sr No	Applied load(N)	Stress in (Mpa)
1	24.5	5.294
2	49.50	10.589
3	73.575	15.884
4	98.1	21.179
5	122.625	26.474
6	147.15	31.769
7	171.675	37.064
8	196.2	42.359
9	220.725	47.654
10	245.25	52.948

III. NUMERICAL ANALYSIS

The Finite Element Method (FEM) is practical application often known as Finite Element Analysis (FEA) is a numerical technique for finding approximate solutions of partial differential equations (PDE) as well as of integral equations. Finite Element Analysis is a simulation technique which evaluates the behavior of components, equipment and structures for various loading conditions including applied forces, pressures and temperatures. Thus, a complex engineering problem with non-standard shape and geometry can be solved using finite element analysis where a closed form solution is not available. The finite element analysis methods result in the stress distribution, displacements and reaction loads at supports etc. for the model. In the finite element analysis 3D model of domestic wind mill blade is developed. After modeling of domestic wind mill blade, the actual supporting boundary conditions are given i.e. fixed support at one end and free at another end (cantilever type). In fixed support there is no any degree of freedom i.e. there is no displacement at any direction. The gradual load is applied on the blade with different load step.

Geometry

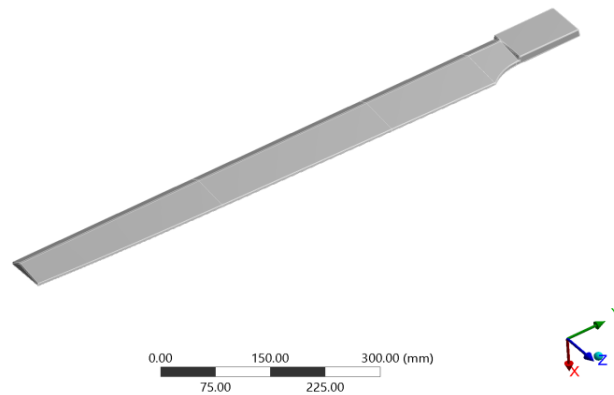


Fig 3.1: 3D Model Of Wind Mill Blade

The meshing of the 3D Model Of Wind Mill Blade having No. of nodes 57566, No. of elements 34146. The load is applied on the blade with different load steps as 24.5N, 49.50N, 73.575 N, 98.1N, 122.15 N, 147.15 N, 171.675 N, 196.2 N, 220.725 N & 245.25 N

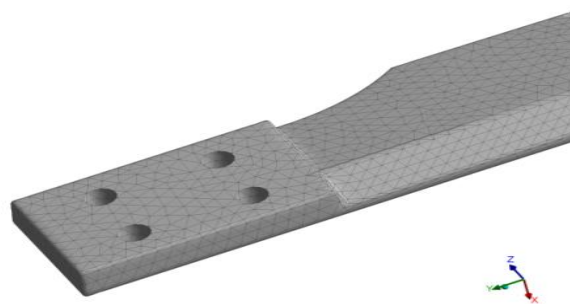


Fig 3.2: Mesh Model Of Wind Mill Blade

Static Structural
Time: 1. s
Fixed Support
Remote Force: 24.525 N



Fig 3.3: Load applied on the wind mill blade

3.1 Numerical results

The numerical results are obtained in the ANSYS software as follows

B: Static Structural Analysis of E-Glass Epoxy windmill Blade
Equivalent Stress
Type: Equivalent (von-Mises) Stress
Unit: MPa
Time: 1

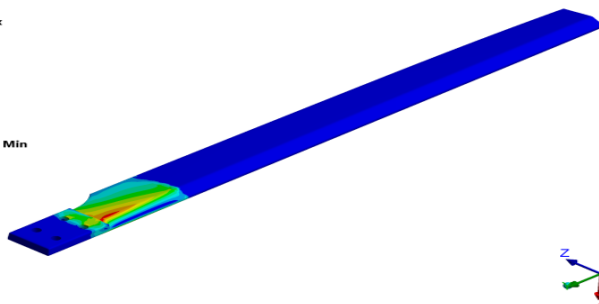
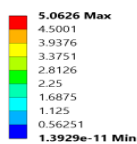


Fig 3.3.1: Numerical Result for Load of 24.5 N

B: Static Structural Analysis of E-Glass Epoxy windmill Blade
Equivalent Stress
Type: Equivalent (von-Mises) Stress
Unit: MPa
Time: 2

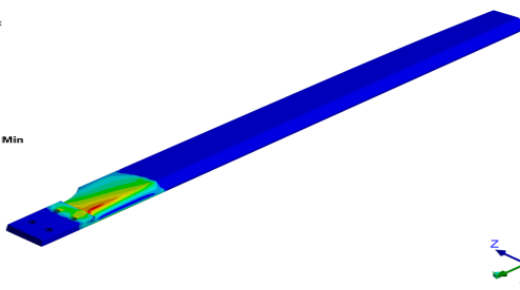
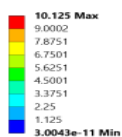


Fig3.3.2: Numerical Result for Load of 49.50 N

B: Static Structural Analysis of E-Glass Epoxy windmill Blade
Equivalent Stress
Type: Equivalent (von-Mises) Stress
Unit: MPa
Time: 3

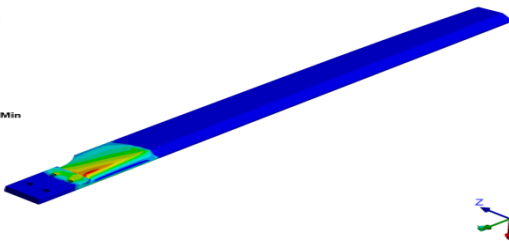
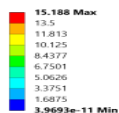


Fig 3.3.3 : Numerical Result for Load of 73.575N

B: Static Structural Analysis of E-Glass Epoxy windmill Blade
Equivalent Stress
Type: Equivalent (von-Mises) Stress
Unit: MPa
Time: 4

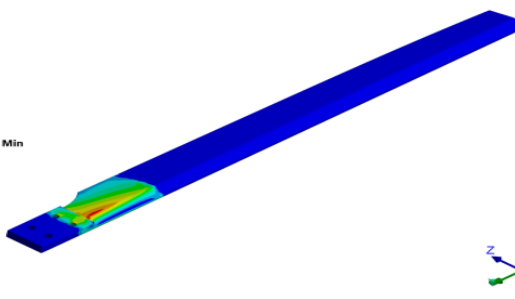
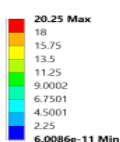


Fig 3.3.4: Numerical Result for Load of 98.1 N

B: Static Structural Analysis of E-Glass Epoxy windmill Blade
Equivalent Stress
Type: Equivalent (von-Mises) Stress
Unit: MPa
Time: 5

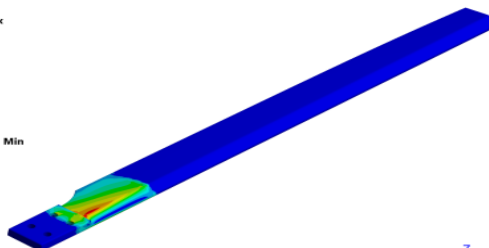
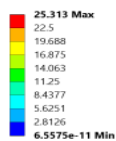


Fig 3.3.5: Numerical Result for Load of 122.15N

B: Static Structural Analysis of E-Glass Epoxy windmill Blade
Equivalent Stress
Type: Equivalent (von-Mises) Stress
Unit: MPa
Time: 6

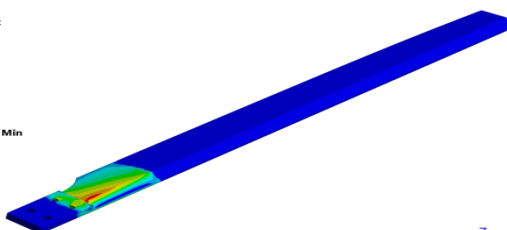
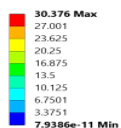


Fig 3.3.6: Numerical Result for Load of 147.15N

B: Static Structural Analysis of E-Glass Epoxy windmill Blade
Equivalent Stress
Type: Equivalent (von-Mises) Stress
Unit: MPa
Time: 7

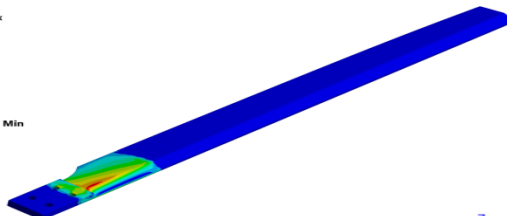
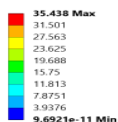


Fig 3.3.7: Numerical Result for Load of 171.675N

B: Static Structural Analysis of E-Glass Epoxy windmill Blade
Equivalent Stress
Type: Equivalent (von-Mises) Stress
Unit: MPa
Time: 8

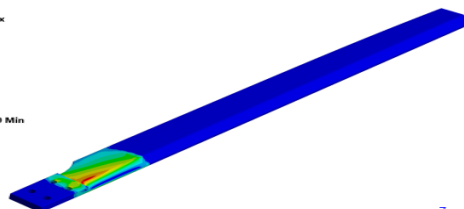
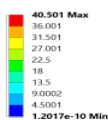


Fig 3.3.8: Numerical Result for Load of 196.2N

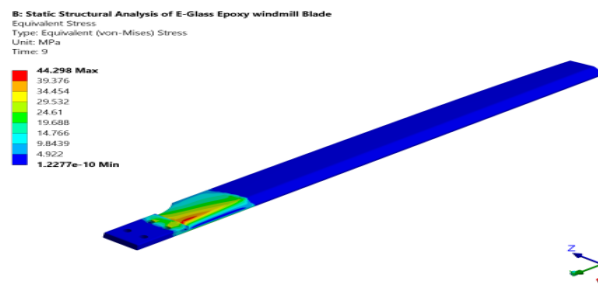


Fig 3.3.9.: Numerical Result for Load of 220.725N

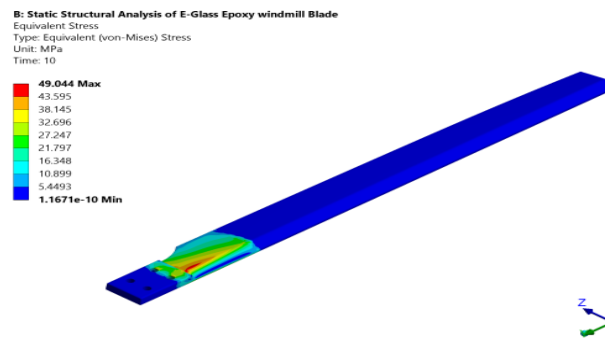


Fig 3.3.10.: Numerical Result for Load of 245.25N

TABLE 2 : Numerical Static Stress Analysis Of Blade

Sr No	Applied load (N)	Numerical stress (Mpa)
1	24.5	5.062
2	49.50	10.125
3	73.575	15.188
4	98.1	20.25
5	122.625	25.313
6	147.15	30.376
7	171.675	35.438
8	196.2	40.501
9	220.725	44.298
10	245.25	49.044

IV.RESULTS AND DISCUSSIONS**TABLE 3: Comparison between theoretical static stresses and numerical static stresses**

Sr No	Applied load (N)	Theoretical stress(Mpa)	Numerical stress (Mpa)
1	24.5	5.294	5.062
2	49.50	10.589	10.125
3	73.575	15.884	15.188
4	98.1	21.179	20.25
5	122.625	26.474	25.313
6	147.15	31.769	30.376
7	171.675	37.064	35.438
8	196.2	42.359	40.501
9	220.725	47.654	44.298
10	245.25	52.948	49.044

From the above results it is seen that there is good conformity between theoretical and simulated static stress of E glass epoxy material domestic wind mill blade. The theoretical calculated stress is compared with numerically obtained values.

V.CONCLUSION

The theoretical static stress is calculated acted on domestic wind mill blade. Also numerically static stress is calculated. The numerical results from finite element analysis show a good conformity between theoretical values.

VII. ACKNOWLEDGEMENT

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