

HUMAN POWERED FLYWHEEL MOTOR BY USING QUICK RETURN RATIO ONE MECHANISM

K. K. Padghan¹, Prof. A.K. Pitale², Prof J. P.Modak³, A.P. Narkhedkar⁴

¹Scholar M.E. (CAD/CAM),PRMIT &R, Badnera,(India)

²Professor,Dept of Mech. Engg. PRMIT.&R,Badnera,(India)

³Emeritus Professor, Dept of Mech Engg ,Priyadarshani C.O.E.Nagpur,(India)

⁴Scholar M.E. (CAD/CAM),PRMIT&R,Badnera,(India)

ABSTRACT

In these days of energy crisis bicycle has remained the only resort as a means of personnel transport in under developed and developing countries. Every effort should be made in improving the performance of the bicycle. There is another reason to improving the performance of the bicycle drive is that, the bicycle drive mechanism is used as a means for converting human energy in the form of mechanical energy in the case of cycle rickshaw and in the case of some manually driven process machines.

Flywheel is used in manually driven machines, mainly as flywheel motor. Flywheel is pedaled to a higher speed to store kinetic energy. This K.E. is drained out in a short period to accomplish a desired process which needs process power beyond human capacities. The pedal operated flywheel provides a wide aspect of living in a different way. It gives us a way to produce energy by the use of human effort.

The following report proposes the use of a pedal operated flywheel to maximize K.E. gain and its optimization. The report firstly defines the problems associated with maximizing the K.E. gain and its use. Subsequent section will compare different types of bicycle mechanism and describe it briefly. The remainder of the report will focus on the optimization of human powered flywheel motor to maximize K.E. gain. To have increased efficiencies, flywheel motors have some special arrangements of inputting power. They are, 1) Quick return ratio one, 2) Elliptical chain wheel, and 3) Double lever inversion. Hence in this paper arrangement and testing values of Quick return ratio one is presented on flywheel motor.

KEY WORDS: - Quick return Ratio-one, Flywheel motor, kinetic energy gain.

I. INTRODUCTION

During 1979-99, Modak J.P. developed a human powered brick making machine for the manufacturing of bricks (Modak J.P. J.P. 1982, 1994, 1997, 1998) [1]. And since then various processes are energized by the human power such as wood turning, cloth washing, chaff cutter [2], potter's wheel, flour mill etc. All these machines are operated by the human power with one common mechanism among them- The Flywheel Motor. The Machine consists of flywheel motor , driven bicycle mechanism with speed increasing gearing , which drives

the shaft of process of process unit through clutch and torque amplification unit (Gupta 1977)[1]. Since ever increasing fuel crises, energy crises, busy schedules of load shading, unemployment justify the need of human powered machines, the constants efforts are being continuously made to optimize the various parameters of these machines so as to provide the ease for the operator and consequently make efficient use of human energy. In an attempt, this paper presents the exhaustive literature survey on the flywheel motor throwing lights on the experimentation done on flywheel motor with double lever inversion for optimizing its performance.

II. FLYWHEEL MOTOR THE CONCEPT

Any machine, to power it by human energy, the maximum power requirement should be 75Watts. Any machine or process requiring more than 75 Watts and if process is intermittent without affecting and product, can also be operated by human energy (Alexandrove 1981)[3]. This is possible with the provision of intermediate energy storing unit which stores the energy of human and supply periodically at required rate to process unit, this is called as “human powered flywheel motor.” Modak J.P. and his associates are working on flywheel motor from 1977. A manually driven brick making machine was first of its kind in which manually energized flywheel motor is used for first time [4]. Essentially the flywheel motor consists of flywheel, which is being driven by a human through a simple bicycle mechanism and pair of speed increasing gears [3]. The schematic of flywheel motor is as shown in fig1.

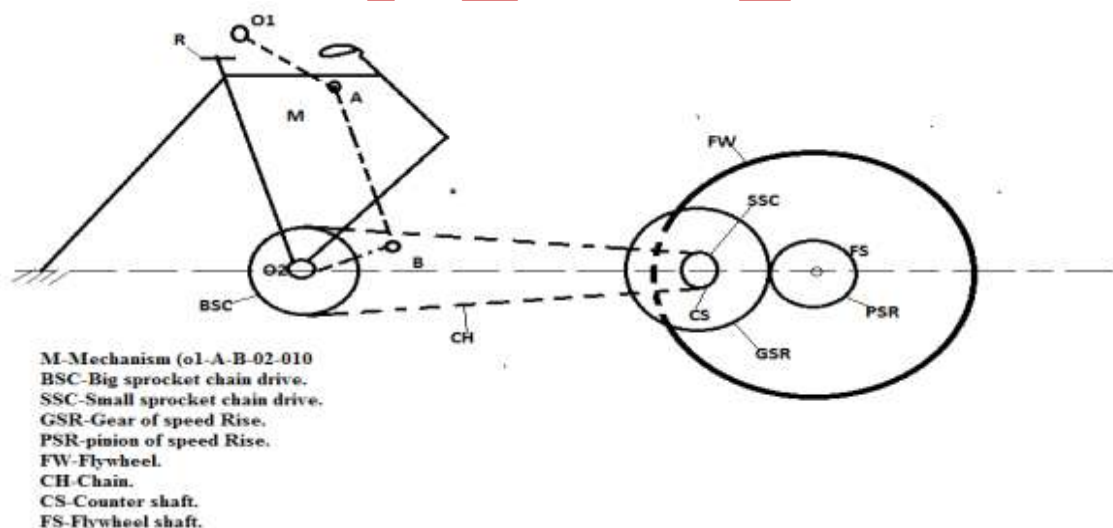


Fig: Schematics of flywheel motor.

A rider pedals the mechanism „M” converting the oscillatory motion of thighs into rotational motion of counter shaft „C”. This countershaft „C” connected to flywheel shaft „FS” with speed increasing transmission consisting of pair of speed gears [4]. Driver pumps the energy in flywheel at energy rate convenient to him [4]. In this way, the muscular energy of human is converted into kinetic energy of flywheel by this man machine and for its efficient use it is necessary to optimize its parameters [4].

III. DESIGN CONSIDERATION IN FLYWHEEL MOTOR.

At the beginning, the flywheel motor was not based on any design data, rather it was built only on the institution of human[4]. Later with the numerous experimentation the design data is made available which is discussed below.

A. MODIFICATION IN EXISTING BICYCLE MECHANISM.

Modak J.P (1985) has established the relationship between the useful torques developed at the crank as function of crank position during its revolution [5]. Modak J.P. also observed that out of 360° rotation of pedal crank, only from 30° - 115° of crank position from top dead center is useful. The rest of the period of crank position i.e. 0° - 30° and 115° - 162° is not effectively used and from 162° - 360° is completely idle. Even when both the cranks are considered the useful driving angle is found to be 154° .[5]. Consequently for maximum utilization of operators energy Modak J.P. suggested three modified mechanisms namely Quick return ratio one, Double lever inversion and Elliptical sprocket[5].Based on his mathematical modeling he concluded improvement of 17%,38%, and 18% in human energy utilization for Quick return ratio one, Double lever inversion and Elliptical sprocket respectively. This performance of various bicycle drives then was experimentally verified by Modak J.P , Chandurkar K.C. et ,al (1987) and found almost matching with theoretical values[6].

B. FLYWHEEL SPEED AND MOMENT OF INERTIA

Modak J.P(1987) during the experimentation has observed the maximum thigh oscillation for the average person of 165 cm stature from age group 20-22 years is 40. [7]. With the available chain drive for existing 22” bicycle frame the flywheel speed of 240 rpm was fair enough from point of total speed rise from pedals to flywheel shaft [7]. Further with calculation Modak J.P.(1987) has determined the size of flywheel with the objective to store the maximum energy irrespective of speed fluctuations(180-240 rpm)[7]. The Flywheel rim diameter is found to 82 cm which gives the weight of flywheel as 150Kg and 266 Kg for 240 rpm and 180 rpm respectively. Hence Modak J.P.(1987) suggested the flywheel with 150 Kg @240 rpm[7]. Further Modak J.P.(1987) has also found that driving torque of pedal is unaffected by increasing flywheel moment of inertia and stores same energy for same frequency of thigh oscillation [7].

C. GEAR RATIO

Modak J.P. (1987) suggested the value of gear ratio as 4:1 so as to reduce the effect of jerk induced at process unit shaft as result of energy or momentum exchange during the clutch engagement. If lower value of gear ratio is to be used then flywheel speed should be maintained higher than 240 rpm [7].

Quick return ratio-one:- From figure

- O_1B is thigh length,
- AB is length,
- O_2A is crank length,
- O_1O_2 is frame.

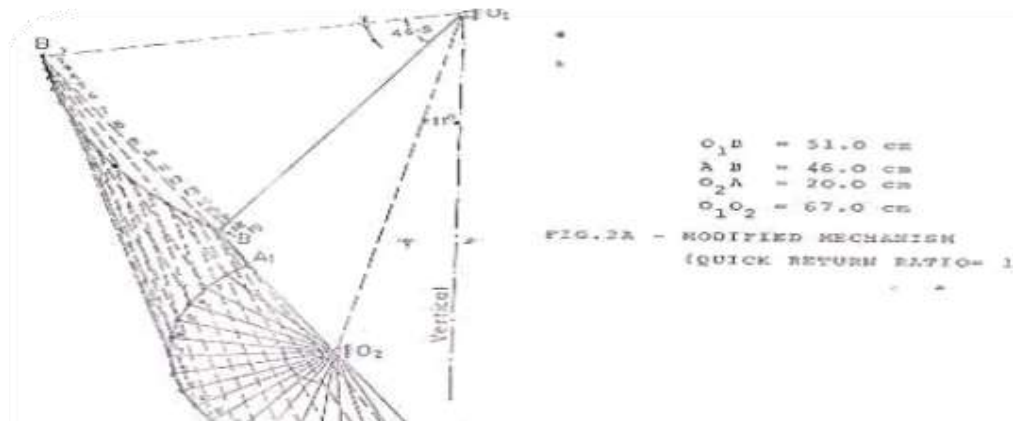
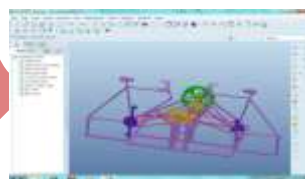


Fig: Modified Mechanism (Quick Return Ratio = 1)

It is modified form of mechanism called as Quick Return Ratio One. In the existing mechanism, the ratio of forward travel to return travel is 0.82. In the Quick Return Ratio One, the ratio is one therefore, the second paddle will be immediately ready when the first one goes down.

In this, the thigh oscillation angle, thigh length and the leg length are kept same. In existing mechanism, the crank length is 18.5 cm and in QRR- one it is 20 cm. Similarly, in existing mechanism the frame length i.e. crank centre to rider's hip joint 74 cm and frame inclination to vertical is 20° . But in QRR-one, the frame length i.e. Crank centre to rider's hip joint 67cm and frame inclination to vertical is 11° .



CAD MODELING OF FABRICATED HUMAN POWERED FLYWHEEL MOTOR WITH QUICK RETURN RATIO ONE BY USING PRO-E



PRACTICAL MODEL OF QUICK RETURN RATIO ONE IN FLYWHEEL MOTOR

IV. THE READINGS AND CALCULATION FOR KINETIC ENERGY DEVELOPED IN TESTS OF ACTUAL SETUP.

When the model of human powered flywheel motor with quick return ratio one was fabricated in our lab, we took readings of kinetic energy for 15 and 30 seconds, with 2nd, 3rd, and top gear. First we did this for weight wise for eight weight groups between 40-80 kg. Then we did this for age wise for eight age groups between 20-60.

Calculations for Kinetic energy stored in flywheel.

Assuming the density for the flywheel material as 7874 kg/m^3 i.e., for cast iron and weight of arm and hub to be 15% more to be multiplied to moment of inertia of flywheel of rim.

The dimensions of the flywheel are as follows:

Outside diameter of flywheel (D_o) = $40\text{cm} = 400\text{mm}$

Rim thickness (h) = $0.05 \text{ Cm} = 5\text{mm}$

Width of the flywheel (b) = $0.6 \text{ Cm} = 60\text{mm}$

The following figure shows the dimensional details of the flywheel.

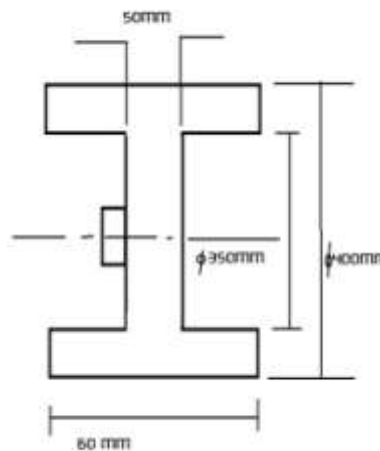


Fig: Dimension of the flywheel

Now the mass of the flywheel is given by,

Mass = volume x density.

Volume of the flywheel is calculated as follows

$$\text{Mass} = [(\pi D^2 \text{Area}) \times [\text{Density}]]$$

$$\text{Mass} = [(\pi D^2 (b \times t)) \times [\text{Density}]]$$

$$\text{Mass} = [(\pi \times 0.4^2 \times (0.06 \times 0.005)) \times [7874]]$$

$$\text{Mass} = 2.968 \text{ Kg. s}$$

$$= 3 \text{ Kg (Approx.)}$$

Now Moment of Inertia of Flywheel (Only Rim)

$$I = w/g*(k^2)$$

$$I = 3/9.81* (0.2^2)$$

$$I = 0.0122 \text{ kg.s.m}^2$$

Consider Weight of Arm and hub.

$$M. I. \text{ Flywheel} = 0.0122*1.15$$

$$= 0.01403$$

$$= 0.015 \text{ kg.s.m}^2 \text{ (Approximately)}$$

Kinetic Energy Stored in Flywheel

$$= \frac{1}{2} I \omega^2$$

$$= \frac{1}{2} * 0.0122 * (2\pi N/60)^2$$

$$K.E. = \frac{1}{2} * 0.0122 * (2\pi/60)^2 * N^2$$

$$K.E. = 6.6894 * 10^{-5} * N^2 \text{ Joules.}$$

Consider RPM = 445 Second Gear,

$$\text{Kinetic Energy (K.E)} = 6.6894 * 10^{-5} * 445^2 \text{ Joules.}$$

$$= 13.24 \text{ Joules.}$$

Consider RPM = 523 Third Gear,

$$\text{Kinetic Energy (K.E)} = 6.6894 * 10^{-5} * 523^2 \text{ Joules.}$$

$$= 18.85 \text{ Joules.}$$

Consider RPM = 558 Top Gear,

$$\text{Kinetic Energy (K.E)} = 6.6894 * 10^{-5} * 558^2 \text{ Joules.}$$

$$= 21.46 \text{ Joules.}$$

Similarly the kinetic energy stored for each case is calculated .

1) AGE WISE:

Sr. No.	Age of Person In Year	Cyclic Time (Sec.)	For load 12 Kg Speed of Flywheel in RPM			Kinetic Energy Gain in Jules		
			2 nd Gear	3 rd Gear	Top Gear	2 nd Gear	3 rd Gear	Top Gear
1	20-25	15	445	523	558	13.24	18.85	21.46
		30	469	530	590	15.48	19.77	24.50
2	25-30	15	305	415	480	6.54	12.12	16.22
		30	317	427	485	7.07	12.83	16.55
3	30-35	15	315	423	510	6.98	12.59	18.31
		30	320	438	523	7.20	13.50	19.25
4	35-40	15	335	442	527	7.90	13.75	19.55

		30	342	438	540	8.23	13.89	20.52
5	40-45	15	347	432	545	8.47	13.13	20.91
		30	352	438	569	8.72	12.24	22.79
6	45-50	15	357	414	534	8.97	12.13	20.97
		30	352	438	569	8.72	13.50	22.79
7	50-55	15	249	294	579	4.36	6.08	23.60
		30	216	310	633	3.28	6.76	28.20
8	55-60	15	223	263	510	3.50	4.86	18.31
		30	212	275	525	3.16	5.32	14.47

Table 1:- Kinetic Energy Gain with respect to age of person.

2) WEIGHT WISE:

Sr. No.	Weight of Person In Kg	Cyclic Time (Sec.)	For load 12 Kg Speed of Flywheel in RPM			Kinetic Energy Gain in Jules		
			2 nd Gear	3 rd Gear	Top Gear	2 nd Gear	3 rd Gear	Top Gear
1	40-45	15	314	370	432	7.02	9.63	13.13
		30	317	362	443	7.07	9.15	13.78
2	45-50	15	357	414	534	8.72	9.15	20.07
		30	352	438	569	8.72	13.50	22.79
3	50-55	15	340	445	520	8.15	13.82	19.20
		30	342	438	538	8.23	13.50	20.47
4	55-60	15	347	432	443	8.47	13.13	13.78
		30	352	417	569	8.72	12.24	22.79
5	60-65	15	440	523	589	13.58	19.25	24.42
		30	460	527	594	14.89	19.55	24.43
6	65-70	15	305	415	480	6.54	12.12	16.22
		30	317	427	485	7.07	12.83	16.55
7	70-75	15	249	295	580	6.54	12.12	23.68
		30	220	310	630	3.27	6.76	27.94
8	75-80	15	445	520	580	13.82	19.20	23.68
		30	462	525	590	15.02	19.45	24.50

Table 2:- Kinetic Energy Gain with respect to weight of person.



V. CONCLUSION



Due to ever increasing energy crises the use of human energized machines are increasing day by day. The numbers of advantages are associated with this such as unavailability of power specially in rural side of India, less skilled operators, unemployment, bicycle exercising etc. Hence human power machines seem to have great future ahead. In this paper the HPFM with Quick return ratio mechanism is proposed. As well as the readings of kinetic energy developed for limited period, weight wise and age wise, are tabulated.

REFERENCES

- [1] Modak J.P “Bicycle and its kinematics and modifica-tions”. National conference mach Mech; February 1985;pp5-11
- [2] Modak J.P, Bapat A.R. “ Improvement in experimen-tal setup for establishing generalized experimental model of various dynamic responses for A manually energised flywheel motor”
- [3] Modak J.P, Bapat A.R. “Various efficiencies of hu-man powered flywheel motor” Human power number volume 54;pp21-23
- [4] Modak J.P “ design and development of human ener-gised chaff cutter”
- [5] Modak J.P , Moghe S.D “Design and development of human powered machine for the manufacture of lime flyash sand bricks” Human power; volume 13 num-ber2;1998;pp3-7.
- [6] Modak J.P. , Chandurkar K.C., Singh M.P, Yadpana-war A.G “Experimental verification of various bi-cycle drive mechanism part1” Proceedings of AMSE conference modeling and simulation Karisurhe west Germeny, july 20-22 1987;pp139-160.

BIOGRAPHIES

	<p>K.K. PADGHAN is M.E. Scholar at Prof. Ram Meghe Institute of technology and research Badnera, Maharashtra, India. He is basically Mechanical Engineer from Prof. Ram Meghe Institute of technology and research Badnera, Maharashtra, India.. Presently he is pursuing his post graduate studies from Ram Meghe Institute of technology and research Badnera, Maharashtra, India. His area of interest is CAD, CAM, Production engineering</p>
	<p>A.K.PITALE is working as professor at Prof. Ram Meghe Institute of technology and research Badnera, Maharashtra, India. He is basically mechanical engineer having its post graduation in Mechanical Engineering. His subjects of interest are Thermal power engineering, CAD, CAM</p>

	<p>J.P.MODAK is working as Emeritus Professor at Dept of Mechanical Engineering, Priyadarshani C.O.E.Nagpur, India</p>
	<p>A.P.NARKHEDKAR is M.E. Scholar at Prof. Ram Meghe Institute of technology and research Badnera, Maharashtra, India. He is basically Mechanical Engineer from Jawaharlal Darda Institute of Technology, Yavatmal, India. Presently he is pursuing his post graduate studies from Ram Meghe Institute of technology and research Badnera, Maharashtra, India. His area of interest is CAD, CAM, Production engineering</p>